

References:

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7. Heat transfer to fluids by forced convection in laminar and turbulent flows

Example:

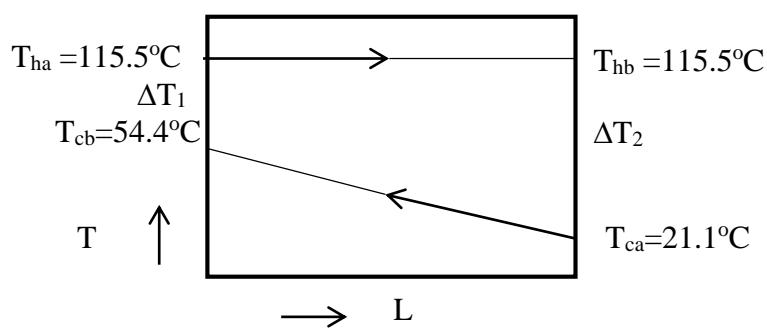
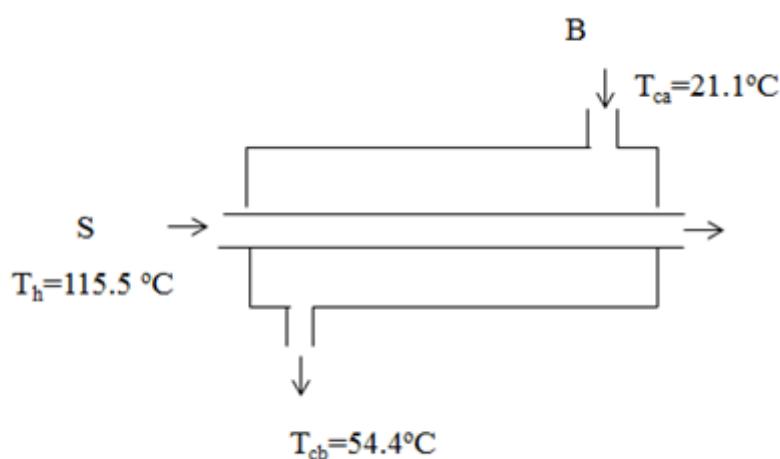


Table 10: Steel pipe dimensions

2 in sch 40 $D_{ii}=2.067 \text{ in} = 52.5 \times 10^{-3} \text{ m}$ $D_{io}=2.375 \text{ in} = 60.32 \times 10^{-3} \text{ m}$

3 in sch 40 $D_{oi}=3.068$ in=77.92x10⁻³ m

$$D_e = D_{oi} - D_{io} = (77.92 - 60.32) \times 10^{-3} \text{ m} = 17.6 \times 10^{-3} \text{ m}$$

$$Re = \frac{D_e v \rho}{\mu} = \frac{(D_{oi} - D_{io}) \dot{m} \rho}{\rho \left[\frac{\pi}{4} D_{oi}^2 - \frac{\pi}{4} D_{io}^2 \right] \mu} = \frac{\dot{m}}{\frac{\pi}{4} (D_{oi} + D_{io}) \mu} = \frac{0.554}{\frac{\pi}{4} (77.92 + 60.32) \times 10^{-3} \times 0.51 \times 10^{-3}} = 10005$$

$Re > 10^4$ Turbulent flow

For turbulent flow; since $\frac{L}{D} > 150$ we ignore the effect of pipe length.

$$\frac{4.572}{17.6 \times 10^{-3}} = 259.7 > 150$$

$$Nu = \frac{h_o D_e}{k} = 0.023(Re)^{0.8}(Pr)^{0.4} \quad (\text{n}=0.4, B \text{ is heating})$$

$$Pr = \frac{C_p \mu}{k}$$

$$h_o = \frac{0.156}{17.6 \times 10^{-3}} \times 0.023 \times (10005)^{0.8} \left(\frac{2424.4 \times 0.51 \times 10^{-3}}{0.156} \right)^{0.4} = 739.8 \frac{W}{m^2 \text{ } ^\circ C}$$

Viscosity value of B is small, so we ignored viscosity correction factor (ϕ_v)

For B; $\mu=0.51 \times 10^{-3}$ kg/(m.s)

$$\Delta T_i = \frac{\frac{1}{h_i}}{\frac{1}{h_i} + \frac{D_i}{D_o} h_o} x \Delta T = \frac{\frac{1}{11338.2}}{\frac{1}{11338.2} + \frac{0.052}{0.0603}} x 739.8 (115.5 - 37.75) = 1.06 \times 10^{-5}$$

For heating $T_w = T + \Delta T_i$

ΔT_i is very low. So $T_w = T$ and viscosity does'nt change.

We neglect viscosity correction factor.

$$q = \dot{m} C_p \Delta T = 0.554 \times 2424.4 \times (54.4 - 21.1) = 44725.8 \frac{J}{s}$$

$$\Delta T_L = \frac{\Delta T_1 - \Delta T_2}{\ln \frac{\Delta T_1}{\Delta T_2}} = \frac{(T_{ha} - T_{cb}) - (T_{hb} - T_{ca})}{\ln \frac{T_{ha} - T_{cb}}{T_{hb} - T_{ca}}}$$

$$\Delta T_L = \frac{(115.5 - 54.4) - (115.5 - 21.1)}{\ln \frac{115.5 - 54.4}{115.5 - 21.1}} = 76.55 \text{ } ^\circ C$$

$$U_o = \frac{q}{A_o \Delta T_L} = \frac{44725.8}{\pi \times 60.32 \times 10^{-3} \times 4.572 \times 76.55} = 674.4 \frac{W}{m^2 {}^\circ C}$$

$$U_o = \frac{1}{\frac{D_o}{D_i h_{d_i}} + \frac{D_o}{D_i h_i} + \frac{x_w}{k_m} \frac{D_o}{D_L} + \frac{1}{h_o} + \frac{1}{h_{d_o}}}$$

$$\overline{D_L} = \frac{D_o - D_i}{\ln \frac{D_o}{D_i}} = \frac{0.0603 - 0.052}{\ln \frac{0.0603}{0.052}} = 0.056 \text{ m}$$

$$D_o = 0.0603 \text{ m}$$

$$D_i = 0.052 \text{ m}$$

$$\overline{D_L} = 0.056 \text{ m}$$

$$x_w = \frac{D_o - D_i}{2} = \frac{0.0603 - 0.052}{2} = 0.0041$$

$k_m = 386 \text{ W/m}^\circ\text{C}$ Copper pure (Table 2)

$$674.4 = \frac{1}{\frac{0.0603}{0.052 \times 11338.2} + \frac{0.0041}{386} \times \frac{0.0603}{0.056} + \frac{1}{739.8} + \frac{1}{h_{d_o}}}$$

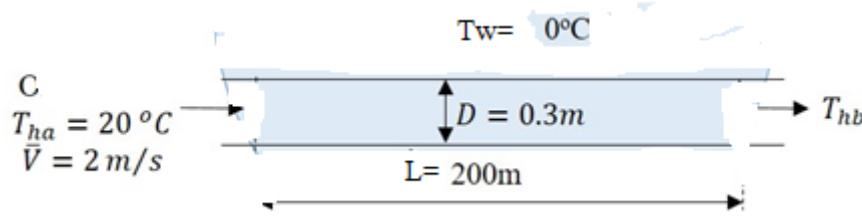
$$674.4 = \frac{1}{0.000102 + 0.000011 + 0.00135 + \frac{1}{h_{d_o}}}$$

$$674.4 = \frac{1}{(0.00146 + \frac{1}{h_{d_o}})}$$

$$(0.00146 + \frac{1}{h_{d_o}}) = \frac{1}{674.4}$$

$$R = \frac{1}{h_{d_o}} = 2.2 \times 10^{-5} \frac{m^2 {}^\circ C}{W}$$

Example :



$$q = \dot{m} C_p (T_{ha} - T_{hb}) = h_i A_i \Delta T_i$$

T_{hb} must be assumed.

1. assumption: $T_{hb} = 10^\circ\text{C}$

$$\bar{T} = \frac{T_{ha} + T_{hb}}{2} = \frac{20 + 10}{2} = 15^\circ\text{C}$$

Properties of engine oil at 15°C

$$\mu = 1.5815 \text{ kg/ms}$$

$$\rho = 890.825 \text{ kg/m}^3$$

$$\rho_{20^\circ\text{C}} = 888.1 \text{ kg/m}^3$$

$$C_p = 1860 \text{ J/kg}^\circ\text{C}$$

$$k = 0.1455 \text{ W/m}^\circ\text{C}$$

$$Pr = 20217$$

$$\mu_w = 3.814 \text{ kg/ms} \quad (\text{at } 0^\circ\text{C})$$

$$Re = \frac{D\bar{V}\rho}{\mu} = \frac{0.3 \text{ m} \times 2 \frac{\text{m}}{\text{s}} \times 890.825}{1.5815 \frac{\text{kg}}{\text{ms}}} = 337.97 < 2100 \quad \text{Laminar flow}$$

$$\frac{L}{D} = \frac{200}{0.3} = 666.67 > 150 \quad \text{fully developed flow}$$

$$Nu = 2 Gz^{1/3} \left(\frac{\mu}{\mu_w} \right)^{0.14} = 2 \left(\frac{\dot{m} \times C_p}{k \times L} \right)^{1/3} \left(\frac{\mu}{\mu_w} \right)^{0.14}$$

Mass flow rate at the beginning of the pipeline.

$$\dot{m} = \rho_{20^\circ\text{C}} \times V \times S = (888.1) \frac{\text{kg}}{\text{m}^3} \times (2) \frac{\text{m}}{\text{s}} \times \left(\frac{\pi}{4} 0.3^2 \right) \text{m}^2 = 125.55 \frac{\text{kg}}{\text{s}}$$

$$Nu = \frac{hD}{k} = \frac{h \times 0.3\text{m}}{0.1455 \frac{\text{W}}{\text{m}^\circ\text{C}}} = 2 \left(\frac{125.55 \frac{\text{kg}}{\text{s}} \times \frac{1860 \text{ J}}{\text{kg}^\circ\text{C}}}{0.1455 \frac{\text{W}}{\text{m}^\circ\text{C}} \times 200\text{m}} \right)^{1/3} \left(\frac{1.5815 \frac{\text{kg}}{\text{ms}}}{3.814 \frac{\text{kg}}{\text{ms}}} \right)^{0.14} = 35.40$$

$$h = 17.169 \frac{\text{W}}{\text{m}^2 \text{ }^\circ\text{C}}$$

$$\dot{m} C_p (T_b - T_a) = h_i \pi D_i L (T_w - \bar{T})$$

$$\left(125.55 \frac{\text{kg}}{\text{s}} \right) \left(1860 \frac{\text{J}}{\text{kg}^\circ\text{C}} \right) (20 - T_{hb}) = \left(17.169 \frac{\text{W}}{\text{m}^2 \text{ }^\circ\text{C}} \right) \pi (0.3 \text{ m}) (200 \text{ m}) \left(\frac{20^\circ\text{C} + T_b}{2} - 0^\circ\text{C} \right)$$

$$T_{hb} = 19.72 \text{ } ^\circ\text{C} \neq 10 \text{ } ^\circ\text{C}$$

2. assumption: $T_{hb} = 19.72 \text{ } ^\circ\text{C}$

$$\bar{T} = \frac{T_{ha} + T_{hb}}{2} = \frac{20 + 19.72}{2} = 19.86 \text{ } ^\circ\text{C}$$

Properties of engine oil at $19.86 \text{ } ^\circ\text{C}$

$$\mu = 0.8374 \text{ kg/ms}$$

$$\rho = 888.1 \text{ kg/m}^3$$

$$C_p = 1881 \text{ J/kg}^\circ\text{C}$$

$$k = 0.1450 \text{ W/m}^\circ\text{C}$$

$$Pr = 10863$$

$$\mu_w = 3.814 \text{ kg/ms} \quad (\text{at } 0^\circ\text{C})$$

$$Re = \frac{D\bar{\rho}}{\mu} = \frac{0.3 \text{ m} \times 2 \frac{m}{s} \times 888.1}{0.8374 \frac{\text{kg}}{\text{ms}}} = 636.33 < 2100 \quad \text{Laminar flow}$$

$$Nu = 2 Gz^{1/3} \left(\frac{\mu}{\mu_w} \right)^{0.14} = 2 \left(\frac{\dot{m} \times C_p}{k \times L} \right)^{1/3} \left(\frac{\mu}{\mu_w} \right)^{0.14}$$

Mass flow rate will not change.

$$Nu = \frac{hD}{k} = \frac{h \times 0.3m}{0.1450 \frac{W}{m^\circ\text{C}}} = 2 \left(\frac{125.55 \frac{kg}{s} \times \frac{1881 \text{ J}}{kg^\circ\text{C}}}{0.1450 \frac{W}{m^\circ\text{C}} \times 200m} \right)^{1/3} \left(\frac{0.8374 \frac{\text{kg}}{\text{ms}}}{3.814 \frac{\text{kg}}{\text{ms}}} \right)^{0.14} = 32.54$$

$$h = 15.72 \frac{W}{m^2 \circ\text{C}}$$

$$\dot{m} C_p (T_b - T_a) = h_i \pi D_i L (T_w - \bar{T})$$

$$\left(125.55 \frac{kg}{s} \right) \left(1881 \frac{J}{kg^\circ\text{C}} \right) (20 - T_{hb}) = \left(15.72 \frac{W}{m^2 \circ\text{C}} \right) \pi (0.3 \text{ m}) (200 \text{ m}) \left(\frac{20^\circ\text{C} + T_b}{2} - 0^\circ\text{C} \right)$$

$$T_{hb} = 19.75 \text{ } ^\circ\text{C} \cong 19.72 \text{ } ^\circ\text{C}$$

Second Way:

1. assumption $T_{hb} = 20 \text{ } ^\circ\text{C}$

$$\bar{T} = \frac{T_{ha} + T_{hb}}{2} = \frac{20 + 20}{2} = 20 \text{ } ^\circ\text{C}$$

Properties of engine oil at $20 \text{ } ^\circ\text{C}$

$$\mu = 0.8374 \text{ kg/ms}$$

$$\rho = 888.1 \text{ kg/m}^3$$

$$C_p = 1881 \text{ J/kg}^\circ\text{C}$$

$$k = 0.1450 \text{ W/m}^\circ\text{C}$$

$$\Pr = 10863$$

$$\mu_w = 3.814 \text{ kg/ms} \quad (\text{at } 0^\circ\text{C})$$

$$Re = \frac{D\bar{V}\rho}{\mu} = \frac{0.3 \text{ m} \times 2 \frac{m}{s} \times 888.1}{0.8374 \frac{\text{kg}}{\text{ms}}} = 636.33 < 2100 \quad \text{Laminar flow}$$

$$\frac{L}{D} = \frac{200}{0.3} = 666.67 > 150 \quad \text{fully developed flow}$$

$$Nu = 2 Gz^{1/3} \left(\frac{\mu}{\mu_w} \right)^{0.14} = 2 \left(\frac{\dot{m} \times C_p}{k \times L} \right)^{1/3} \left(\frac{\mu}{\mu_w} \right)^{0.14}$$

Mass flow rate at the beginning of the pipeline.

$$\dot{m} = \rho_{20^\circ\text{C}} \times v \times S = (888.1) \frac{\text{kg}}{\text{m}^3} \times (2) \frac{\text{m}}{\text{s}} \times \left(\frac{\pi}{4} 0.3^2 \right) \text{m}^2 = 125.55 \frac{\text{kg}}{\text{s}}$$

$$Nu = 2 Gz^{1/3} \left(\frac{\mu}{\mu_w} \right)^{0.14} = 2 \left(\frac{\dot{m} \times C_p}{k \times L} \right)^{1/3} \left(\frac{\mu}{\mu_w} \right)^{0.14}$$

$$Nu = \frac{hD}{k} = \frac{h \times 0.3m}{0.1450 \frac{W}{m^\circ\text{C}}} = 2 \left(\frac{125.55 \frac{\text{kg}}{\text{s}} \times \frac{1881 \text{j}}{\text{kg}^\circ\text{C}}}{0.1450 \frac{W}{m^\circ\text{C}} \times 200\text{m}} \right)^{\frac{1}{3}} \left(\frac{0.8374 \frac{\text{kg}}{\text{ms}}}{3.814 \frac{\text{kg}}{\text{ms}}} \right)^{0.14} = 32.54$$

$$h = 15.72 \frac{W}{m^2 \circ\text{C}}$$

$$\dot{m} C_p (T_b - T_a) = h_i \pi D_i L (T_w - \bar{T})$$

$$\left(125.55 \frac{\text{kg}}{\text{s}} \right) \left(1881 \frac{\text{j}}{\text{kg}^\circ\text{C}} \right) (20^\circ\text{C} - T_{hb}) = \left(15.72 \frac{W}{m^2 \circ\text{C}} \right) \pi (0.3 \text{ m}) (200 \text{ m}) \left(\frac{20^\circ\text{C} + T_b}{2} - 0^\circ\text{C} \right)$$

$$T_{hb} = 19.75^\circ\text{C} \cong 20^\circ\text{C}$$

b)

$$q = \dot{m} C_p (T_b - T_a) = \left(125.55 \frac{\text{kg}}{\text{s}} \right) \left(1881 \frac{\text{j}}{\text{kg}^\circ\text{C}} \right) (20^\circ\text{C} - 19.75^\circ\text{C}) = 59040 \text{W} \cong 59 \text{kW}$$