

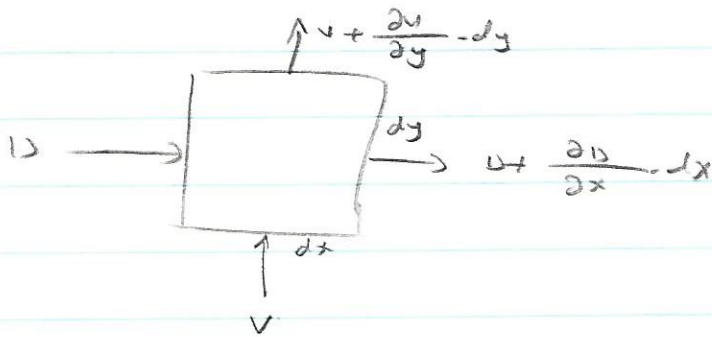
Continuity Equation

In steady flow, amount of mass within the control volume remains constant.

$$\left\{ \begin{array}{l} \text{rate of mass flow} \\ \text{into the control volume} \end{array} \right\} = \left\{ \begin{array}{l} \text{rate of mass flow} \\ \text{out of the control volume} \end{array} \right\}$$

mass flow rate, $\rho \cdot \bar{v} \cdot A$

\downarrow average velocity ↪ cross-sectional area normal to the flow



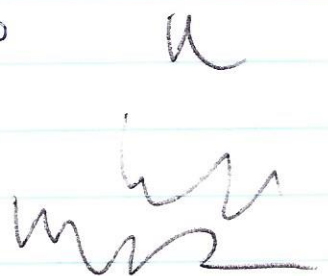
$$\frac{1}{dx \cdot dy} \left[\rho \cdot U \cdot (dy \cdot 1) + \rho v \cdot (dx \cdot 1) = \rho \left(U + \frac{\partial U}{\partial x} \cdot dx \right) (dy \cdot 1) + \rho \left(v + \frac{\partial v}{\partial y} \cdot dy \right) (dx \cdot 1) \right]$$

$$\frac{\rho U}{dx} + \frac{\rho v}{dy} = \frac{\rho U}{dx} + \frac{\partial U}{\partial x} + \frac{\rho v}{dy} + \frac{\partial v}{\partial y}$$

$$\frac{\partial U}{\partial x} + \frac{\partial v}{\partial y} = 0 \quad \leftarrow \text{conservation of mass}$$

In the case of an axially symmetric flow

$$\frac{\partial U}{\partial x} + \frac{1}{r} \cdot \frac{\partial}{\partial r} (r v) = 0$$



Momentum Equations

Newton's 2nd law. expression of momentum balance
net force acting on the control volume is equal to
the mass \times acceleration of the fluid within
the control volume.

$$\sum m \cdot a_x = F_{\text{surface},x} + F_{\text{body},x}$$

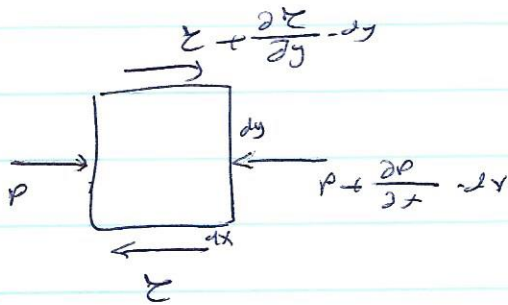
$$\sum m = \rho (dx \cdot dy \cdot d)$$

$$U = U(x, y)$$

$$dU = \frac{\partial U}{\partial x} \cdot dx + \frac{\partial U}{\partial y} \cdot dy$$

$$a_x = \frac{dU}{dt} = \frac{\partial U}{\partial x} \cdot \frac{dx}{dt} + \frac{\partial U}{\partial y} \cdot \frac{dy}{dt}$$
$$= u \frac{\partial U}{\partial x} + v \frac{\partial U}{\partial y}$$

The forces acting on a surface are due to pressure
viscous effects



$$F_{\text{Surface},x} = \left\{ p - \left(p + \frac{\partial p}{\partial x} dx \right) \int_0^{dy} \left[\tau - \left(\tau + \frac{\partial \tau}{\partial y} dy \right) \right] dy \right\} (dx \cdot 1)$$
$$= \left(\frac{\partial \tau}{\partial y} dy - \frac{\partial p}{\partial x} dx \right) (dx \cdot dy \cdot 1)$$

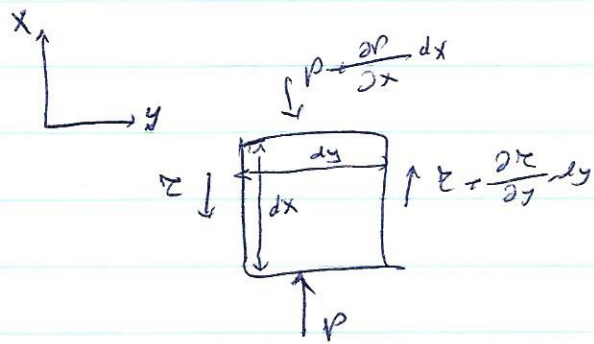
$$\tau = \mu \frac{dU}{dy}$$

$$\rho m \cdot dx = F_{\text{surface}, x} + F_{\text{body}, x}$$

$$\rho(dx \cdot dy \cdot 1) \cdot \left(u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) = \left(\mu \frac{\partial^2 u}{\partial y^2} - \frac{\partial p}{\partial x} \right) \cdot (dx \cdot dy \cdot 1)$$

$$\rho \left(u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) = \mu \frac{\partial^2 u}{\partial y^2} - \frac{\partial p}{\partial x}$$

In y-direction for natural convection,



$$\rho m \cdot dx = F_x$$

$$dx = u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y}$$

$$F_x = \left\{ p - \left(p + \frac{\partial p}{\partial x} dx \right) \right\} (dy \cdot 1) - \left\{ z - \left(z + \frac{\partial z}{\partial y} dy \right) \right\} (dx \cdot 1) - \rho g (dx \cdot dy \cdot 1)$$

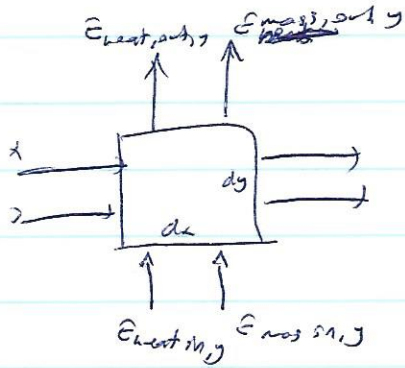
$$F_x = \frac{\partial p}{\partial x} dy \cdot dx \cdot 1 - \frac{\partial z}{\partial y} dx \cdot dy \cdot 1 - \rho g (dx \cdot dy \cdot 1)$$

$$z = \mu \frac{dy}{dy} \Rightarrow$$

$$\rho \left(u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) = \mu \frac{\partial^2 u}{\partial y^2} - \frac{\partial p}{\partial x} - \rho g$$

Conservation of Energy Equation

$$(\dot{E}_{in} - \dot{E}_{out})_{\text{by heat}} + (\dot{E}_{in} - \dot{E}_{out})_{\text{by work}} + (\dot{E}_{in} - \dot{E}_{out})_{\text{by mass}} = 0$$



$$(\dot{E}_{in} - \dot{E}_{out})_{\text{by mass, x}} = (\dot{m}h)_x - \left((\dot{m}h)_x + \frac{\partial(\dot{m}h)}{\partial x} dx \right)$$

h: Enthalpy.

$$= - \frac{\partial(\dot{m}h)}{\partial x} dx$$

$$= - \frac{\partial}{\partial x} (\rho U \cdot (dy \cdot l) - C_p T) \cdot dx$$

$$= - \rho C_p \cdot \left(U \frac{\partial T}{\partial x} + T \frac{\partial U}{\partial x} \right) (dx \cdot dy)$$

$$(\dot{E}_{in} - \dot{E}_{out})_{\text{by mass, y}} = (\dot{m}h)_y - \left((\dot{m}h)_y + \frac{\partial(\dot{m}h)}{\partial y} dy \right)$$

$$= - \frac{\partial}{\partial y} (\dot{m}h) dy$$

$$= - \frac{\partial}{\partial y} (\rho V \cdot (dx \cdot l) - C_p T)$$

$$= - \rho C_p \cdot \left(V \frac{\partial T}{\partial y} + T \frac{\partial V}{\partial y} \right) \cdot (dx \cdot dy)$$

$$(\dot{E}_{in} - \dot{E}_{out})_{\text{by mass}} = \left\{ - \rho C_p \left(U \frac{\partial T}{\partial x} + T \frac{\partial U}{\partial x} \right) - \rho C_p \left(V \frac{\partial T}{\partial y} + T \frac{\partial V}{\partial y} \right) \right\} dx dy$$

$$\begin{aligned}
 (\dot{E}_{in} - \dot{E}_{out})_{by\ mass} &= \left\{ -\rho C_p \left(u \frac{\partial T}{\partial x} + T \frac{\partial u}{\partial x} + v \frac{\partial T}{\partial y} + T \frac{\partial v}{\partial y} \right) \right\} dx dy \\
 &= -\rho C_p \left(u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} + \underbrace{T \left(\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} \right)}_{\substack{= \\ 0 \\ \text{continuity equation}}} \right) dx dy
 \end{aligned}$$

$$\begin{aligned}
 (\dot{E}_{in} - \dot{E}_{out})_{by\ mass} &\Rightarrow -\rho C_p \left(u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} \right) dx dy \\
 (\dot{E}_{in} - \dot{E}_{out})_{by\ heat} &= k \left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right) dx dy
 \end{aligned}
 \quad \Rightarrow$$

$$(\dot{E}_{in} - \dot{E}_{out})_{by\ mass} + (\dot{E}_{in} - \dot{E}_{out})_{by\ heat} = 0 \Rightarrow$$

$$\rho C_p \left(u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} \right) = k \left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right)$$

$$\rho C_p \left[\frac{\partial T}{\partial t} + u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} + w \frac{\partial T}{\partial z} \right] = k \left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} \right)$$